





# **Revision history**

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Date	Changed	Rev
November 2017	Dimenions section changes.	0803
August 2017	Added informational note to Technical Data section.	0802
March 2016	Pressure-compensated PVB, open center PVP; Characteristics for float position main spools.	0801
March 2016	Updated for Engineering Tomorrow design.	0710
August 2015	PVPX modules description updated	0709
February 2006 - Aug 2015	Various changes	BA - HH
January 2005	New Edition	AA

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PVG is a hydraulic, load-sensing proportional valve, designed for optimal machine performance and maximum design flexibility.

The PVG valve design is based on a modular concept that enables machine designers to specify a valve solution suitable for multiple market segments across multiple applications.

The load independent proportional control valve and high performance actuator technology combined with a low pressure drop design improves the machine performance and efficiency – increasing productivity and reducing energy consumption.

#### Features of PVG 32

- Load-independent flow control:
  - Oil flow to an individual function is independent of the load pressure of this function
  - Oil flow to one function is independent of the load pressure of other functions
- Reliable regulation characteristics across the entire flow range
- Load sense relief valves for A and B port enables reduced energy loss at target pressure
- Several options for connection threads and flange mount
- Compact design, easy installation and serviceability
- Energy-saving
- Up to 12 basic modules per valve group
- · Low weight





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## **PVG** general description

#### **PVG** modules

#### PVP, pump side modules overview

- Built-in pressure relief valve
- Pressure gauge connection
- Versions:
  - Open center version for systems with fixed displacement pumps
  - Closed center version for systems with variable displacement pumps
  - Pilot oil supply for electrical actuator built into the pump side module
  - Pilot oil supply for hydraulic actuation built into the pump side module
  - Versions prepared for electrical LS unloading valve PVPX

#### PVB, basic modules

- Interchangeable spools
- Depending on requirements the basic module can be supplied with:
  - Integrated pressure compensator in channel P
  - Load holding check valve in channel P
  - Shock/suction valves for A and B ports
  - LS pressure limiting valves individually adjustable for ports A and B
  - Different interchangeable spool variants
  - All versions suitable for mechanical, hydraulic and electrical actuation

#### **Actuation modules**

The basic module is always fitted with mechanical actuator PVM and PVMD, which can be combined with the following as required:

- Electrical actuator (11 32 V ===):
  - PVES proportional, Super
  - PVEH proportional, High performance
  - PVEH-F proportional high performance, Float
  - PVEA proportional low hysteresis
  - PVEM proportional, Medium performance
  - \_ PVEO ON/OFF
  - PVEH-U/PVES-U proportional, voltage control, 0-10 V
  - PVED-CC Digital CAN controlled J1939/ISOBUS
  - PVED-CX Digital CAN controlled CANopen X-tra safety
  - PVEP PWM voltage controlled (11-32 V)
  - PVHC High Current actuator for PVG
- PVMR, cover for Mechanical detent
- PVMF, cover for Mechanical Float
- PVH, cover for Hydraulic actuation

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## **Remote control units**

- Electrical remote control units:
  - PVRE, PVRET
  - PVREL
  - PVRES
  - Prof 1
  - Prof 1 CIP
  - JS120
- Hydraulic remote control unit: PVRHH

- JS6000 - JS7000

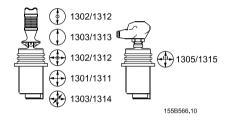
- JS2000

- JS1000 Ball grip

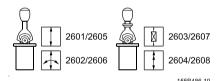
- JS1000 PRO grip

Electrical and hydraulic remote control units

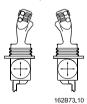
PVRE, electrical control unit, 162F...



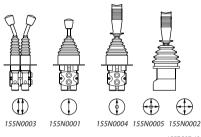
PVREL, electrical control unit, 155U...



Prof 1, 162F...

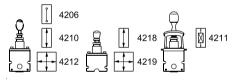


PVRH, hydraulic control unit, 155N...



155B567.10

PVRES, electrical control unit, 155B...





## **Open center PVP**

Description of the example: PVG 32 with open center PVP (fixed displacement pump) and PVB with flow control spool.

When the pump is started and the main spools in the individual basic modules (11) are in the neutral position, oil flows from the pump, through connection P, across the pressure adjustment spool (6) to tank. The oil flow led across the pressure adjustment spool determines the pump pressure (stand-by pressure).

When one or more of the main spools are actuated, the highest load pressure is fed through the shuttle valve circuit (10) to the spring chamber behind the pressure adjustment spool (6), and completely or partially closes the connection to tank to maintain pump pressure. Pump pressure is applied to the right-hand side of the pressure adjustment spool.

The pressure relief valve (1) will open when the load pressure exceed the set value, diverting pump flow back to tank.

In a pressure-compensated basic module the compensator (14) maintains a constant pressure drop across the main spool – both when the load changes and when a module with a higher load pressure is actuated.

With a non pressure-compensated basic module incorporating a load drop check valve (18) in channel P, the check valve prevents return oil flow.

The basic module can be supplied without the load drop check valve in channel P for functions with overcenter valves.

The shock valves PVLP (13) with fixed setting and the suction valves PVLA (17) on ports A and B are used for the protection of the individual working function against overload and/or cavitation.

An adjustable LS pressure limiting valve (12) can be built into the A and B ports of pressure-compensated basic modules to limit the pressure from the individual working functions. The LS pressure limiting valves save energy compared with:

- PVLP all the oil flow to the working function will be led across the combined shock and suction valves to tank if the pressure exceeds the fixed setting.
- LS pressure limiting valves an oil flow of about 2 l/min [0.5 US gal/min] will be led across the LS
  pressure limiting valve to tank if the pressure exceeds the valve setting.

Please see the sectional drawing Sectional view on page 9 for better understanding of this example.

### **Closed center PVP**

Description of the example: PVG 32 with closed center PVP (variable displacement pump) and PVB with flow control spool.

In the closed center version of PVP an orifice (5) and a plug (7) have been fitted instead of the plug (4). This means that the pressure adjustment spool (6) will open to tank when the pressure in channel P exceeds the set value of the pressure relief valve (1). The pressure relief valve in PVP should be set at a pressure of approx. 30 bar [435 psi] above maximum system pressure (set on the pump or external pressure relief valve).

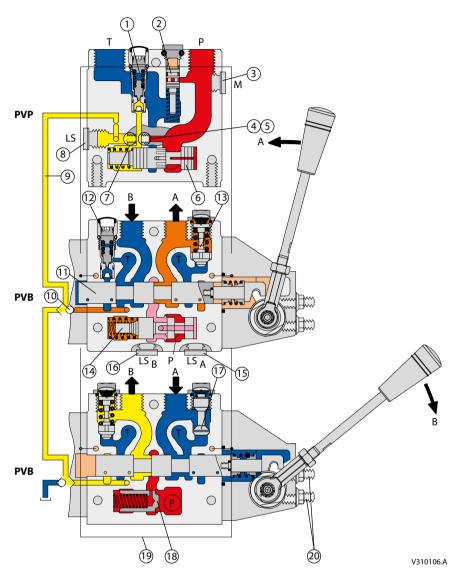
In load sensing systems the load pressure is led to the pump control via the LS connection (8). In the neutral position the pump load sense control sets the displacement so that leakage in the system is compensated, to maintain the set stand-by pressure. When a main spool is actuated the pump load sense control will adjust the displacement so that the set differential pressure (margin) between P and LS is maintained.

Please see the sectional drawing *Sectional view* on page 9 for better understanding of this example.

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# **Sectional view**



- 1. Pressure relief valve
- 2. Pressure reduction valve for pilot oil supply
- 3. Pressure gauge connection
- 4. Plug, open center
- 5. Orifice, closed center
- 6. Pressure adjustment spool
- 7. Plug, closed center
- 8. LS connection
- 9. LS signal
- 10. Shuttle valve

- 11. Main spool
- 12. LS pressure limiting valve
- 13. Shock and suction valve, PVLP
- 14. Pressure compensator
- 15. LS connection, port A
- 16. LS connection, port B
- 17. Suction valve, PVLA
- 18. Load drop check valve
- 19. Pilot oil supply for PVE
- 20. Maximum oil flow adjustment screws for A/B ports



### Load sensing for variable displacement pump supply

The pump receives fluid directly from the reservoir through the inlet line. A screen in the inlet line protects the pump from large contaminants.

The pump outlet feeds directional control valves such as PVG-32, hydraulic integrated circuits (HIC), and other types of control valves.

The PVG valve directs and controls pump flow to cylinders, motors and other work functions. A heat exchanger cools the fluid returning from the valve. A filter cleans the fluid before it returns to the reservoir.

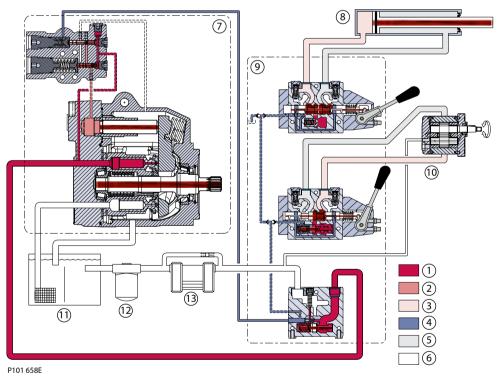
Flow in the circuit determines the speed of the actuators. The position of the PVG valve spool determines the flow demand. A hydraulic pressure signal (LS signal) communicates demand to the pump control.

The pump control monitors the pressure differential between pump outlet and the LS signal, and regulates servo pressure to control the swashplate angle. Swashplate angle determines pump flow.

Actuator load determines system pressure. The pump control monitors system pressure and will decrease the swashplate angle to reduce flow if system pressure reaches the pump control setting.

A secondary system relief valve in the PVG valve acts as a back-up to control system pressure.

## Pictorial circuit diagram



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All types of control valves (incl. proportional valves) can fail, thus the necessary protection against the serious consequences of function failure should always be built into the system. For each application an assessment should be made for the consequences of pressure failure and uncontrolled or blocked movements. To determine the degree of protection that is required to be built into the application, system tools such an FMEA (Failure Mode and Effect Analysis) and Hazard and Risk Analysis can be used.

### **FMEA - IEC EN 61508**

FMEA (Failure Mode and Effect Analysis) is a tool used for analyzing potential risks. This analytical technique is utilized to define, identify, and prioritize the elimination or reduction of known and/or potential failures from a given system before it is released for production. Please refer to the standard IEC FMEA 61508.

### Hazard and risk analysis ISO 12100-1/14121

This analysis is a tool used in new applications as it will indicate whether there are special safety considerations to be met according to the machine directives EN 13849. Dependent on the determined levels conformity this analysis will detirmine if any extra requirements for the product design, development process, production process or maintenance, example the complete product life cycle.



### Warning

All brands and all types of directional control or proportional valves, which are used in many different operation conditions and applications, can fail and cause serious damage. Analyze all aspects of the application. The machine builder/system integrator alone is responsible for making the final selection of the products and assuring that all performance, safety and warning requirements of the application are met. The process of choosing the control system and safety levels is governed by the machine directives EN 13849 (Safety related requirements for control systems).

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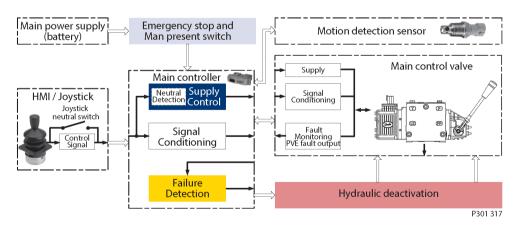
# Example of a control system for manlift

Example of a control system for man-lift



Example of a control system for man-lift using PVE Fault monitoring input signals and signals from external sensors to ensure the PLUS+1° main controllers correct function of the man-lift.

## Typical PVE wiring block diagram



## A

# Warning

It is the responsibility of the equipment manufacturer that the control system incorporated in the machine is declared as being in conformity with the relevant machine directives.

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# PVG 32 – used in system with fixed displacement pumps:

- PVSK, commonly used in crane application full flow dump
- PVPX, LS dump to tank

## PVG 100 - alternative LS dump/pilot supply disconnect:

- PVPP, pilot oil supply shut off
- External cartridge valve connecting LS pressure or main pressure to tank

## PVG 120 – pump disconnect/block for variable pumps:

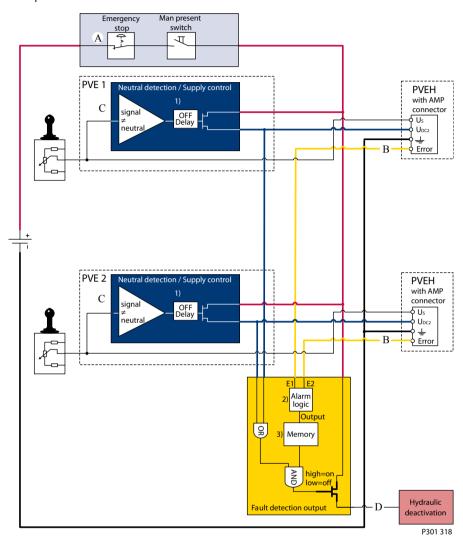
- PVPE, full flow dump for the PVG 120
- External cartridge valve connecting LS pressure to tank

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# **Examples of wiring block diagram**

## Example 1



Typical wiring block diagram using PVEH with neutral power off switch and fault monitoring output for hydraulic deactivation.

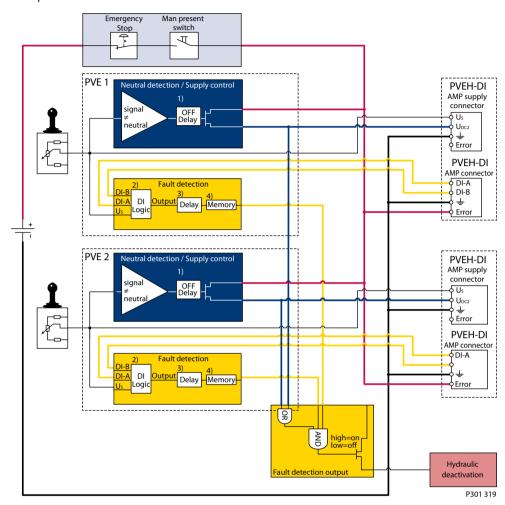
- A Emergency stop / man present switch
- **B** PVE Fault monitoring signals
- **C** Neutral signal detection
- **D** Deactivation of the hydraulic system (System Control Logic, example: PLUS+1® for signal monitoring and triggering signal)

## Warning

It is the responsibility of the equipment manufacturer that the control system incorporated in the machine is declared as being in conformity with the relevant machine directives.



Example 2



Fault monitoring for deactivation of the hydraulic system with extra fault inputs using the PVE's with DI (Direction Indication) function. System Control Logic, example PLUS+1° for signal monitoring and triggering signal for deactivation of the hydraulic system.



## Warning

It is the responsibility of the equipment manufacturer that the control system incorporated in the machine is declared as being in conformity with the relevant machine directives.

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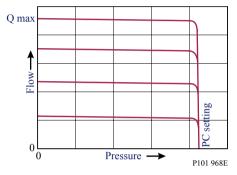


There is a wide range of pressure settings and fluid flow controls.

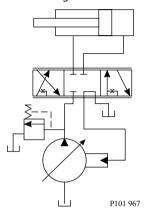
#### Load sensing controls

The LS control matches system requirements for both pressure and flow in the circuit regardless of the working pressure. Used with a closed center control valve, the pump remains in low-pressure standby mode with zero flow until the valve is opened. The LS setting determines standby pressure.

Typical operating curve



Load sensing circuit



Most load sensing systems use parallel, closed center, control valves with special porting that allows the highest work function pressure (LS signal) to feed back to the LS control.

*Margin pressure* is the difference between system pressure and the LS signal pressure. The LS control monitors margin pressure to read system demand. A drop in margin pressure means the system needs more flow. A rise in margin pressure tells the LS control to decrease flow.

## LS control with bleed orifice (do not use with PVG valves)

The load sense signal line requires a bleed orifice to prevent high-pressure lockup of the pump control. Most load-sensing control valves include this orifice. An optional internal bleed orifice is available, for use with control valves that do not internally bleed the LS signal to tank.

### **Integral PC function**

The LS control also performs as a PC control, decreasing pump flow when system pressure reaches the PC setting. The pressure compensating function has priority over the load sensing function.

For additional system protection, install a relief valve in the pump outlet line.

### Load sensing system characteristics:

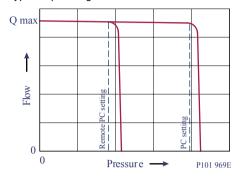
- · Variable pressure and flow
- Low pressure standby mode when flow is not needed
- System flow adjusted to meet system requirements
- Lower torque requirements during engine start-up
- Single pump can supply flow and regulate pressure for multiple circuits
- Quick response to system flow and pressure requirements



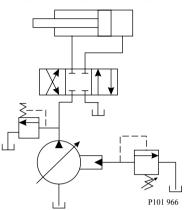
#### Remote pressure compensated controls

The remote PC control is a two-stage control that allows multiple PC settings. Remote PC controls are commonly used in applications requiring low and high pressure PC operation.

Typical operating curve



Closed center circuit with remote PC



The remote PC control uses a pilot line connected to an external hydraulic valve. The external valve changes pressure in the pilot line, causing the PC control to operate at a lower pressure. When the pilot line is vented to reservoir, the pump maintains pressure at the load sense setting.

When pilot flow is blocked, the pump maintains pressure at the PC setting. An on-off solenoid valve can be used in the pilot line to create a low-pressure standby mode. A proportional solenoid valve, coupled with a microprocessor control, can produce an infinite range of operating pressures between the low pressure standby setting and the PC setting.

Size the external valve and plumbing for a pilot flow of 3.8 l/min [1 US gal/min]. For additional system protection, install a relief valve in the pump outlet line.

#### Remote pressure compensated system characteristics:

- Constant pressure and variable flow
- High or low pressure standby mode when flow is not needed
- · System flow adjusts to meet system requirements
- Single pump can provide flow to multiple work functions
- Quick response to system flow and pressure requirements

### Typical applications for remote pressure compensated systems:

- Modulating fan drives
- · Anti-stall control with engine speed feedback
- Front wheel assist
- Road rollers
- · Combine harvesters
- Wood chippers

# PVPC with check valve for open center PVP

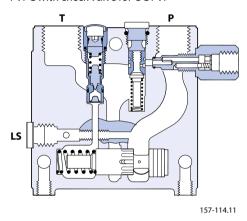
PVPC with check valve is used in systems where it is necessary to operate the PVG 32 valve by means of the electrical remote control without pump flow. When the external solenoid valve is opened, oil from the pressure side of the cylinder is fed via the PVPC through the pressure reducing valve to act as the



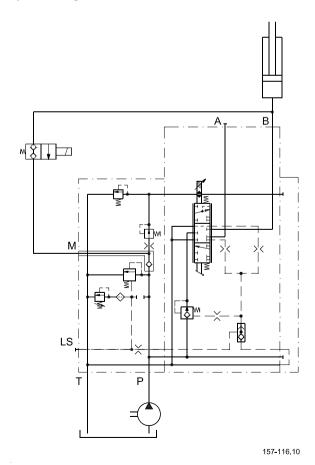
pilot supply for the electrical actuators. This means that a load can be lowered by means of the remote control lever without starting the pump.

The built-in check valve prevents the oil from flowing via the pressure adjustment spool to tank. With the pump functioning normally the external solenoid valve is closed to ensure that the load is not lowered due to the pilot supply oil flow requirement of approximately 1 l/min [0.25 US gal/min]. With closed center PVP the external pilot oil supply can be connected to the pressure gauge connection without the use of a PVPC plug.

PVPC with check valve for OC PVP



Hydraulic diagram



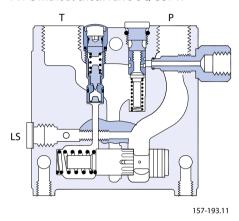


# PVPC without check valve for open or closed center PVP

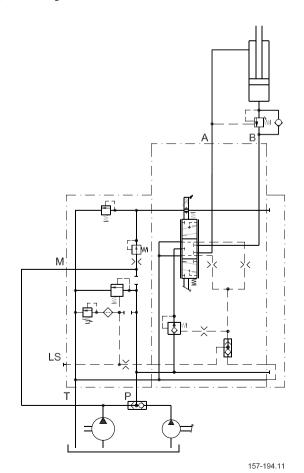
PVPC without check valve is used in systems where it is necessary to supply the PVG 32 valve with oil from a manually operated emergency pump without directing oil flow to the pilot oil supply (oil consumption about 0.5 l/min) [0.13 US gal/min].

When the main pump is working normally, the oil is directed through the PVPC plug via the pressure reduction valve to the electrical actuators.

### PVPC without check valve OC/CC PVP



Hydraulic diagram



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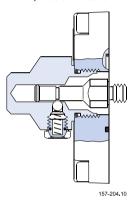
When the main pump flow fails, the external shuttle valve ensures that the oil flow from the manually operated emergency pump is used to pilot open the over center valve and lower the load. The load can only be lowered using the mechanical operating lever of the PVG 32 valve.

### **PVMR**, friction detent

The friction detent PVMR allows the directional spool to be held in any position, resulting in infinitely variable, reversible, pressure compensated flow.

This can be sustained indefinitely without having to continue to hold the mechanical lever. Friction detent spool position may be affected by high differential actuator flow forces and system vibration resulting in work function flow reduction.

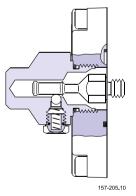
#### PVMR, friction detent



### PVMF, mechanical float position lock

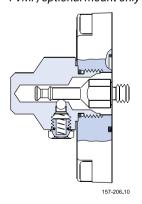
Allows the float spool to be held in the float position after release of the mechanical handle.

PVMF, standard mount only



 $P \rightarrow A \rightarrow F$  (Push-in)

PVMF, optional mount only



 $P \rightarrow A \rightarrow F$  (Pull-out)

## PVBS, main spools for flow control (standard)

When using standard flow control spools, the pump pressure is determined by the highest load pressure. This is done either via the pressure adjustment spool in open center PVP (fixed displacement pumps) or via the pump control (variable displacement pumps).

In this way the pump pressure will always correspond to the load pressure plus the stand-by pressure of the pressure adjustment spool or the pump control. This will normally give optimum and stable adjustment of the oil flow.



# PVBS, main spools for flow control (linear characteristic)

PVBS main spools with linear characteristic have less dead band than standard spools and a proportional ratio between control signal and oil flow in the range beyond the dead band. PVBS with linear characteristic must never be used together with PVEM electrical actuators.

The interaction between the small dead band of the spools and the hysteresis of the PVEM actuator of 20% involves a risk of building up a LS pressure in neutral position.

In a few systems load sensing pump pressure may result in unstable adjustment of the oil flow and a tendency towards system hunting.

This may be the case with working functions that have a large moment of inertia or over-center valves. In such systems main spools for pressure control can be advantageous.

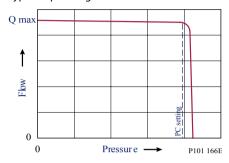
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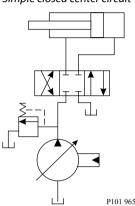
### PVBS, main spools with pressure compensated control

The PC control maintains constant system pressure in the hydraulic circuit by varying the output flow of the pump. Used with a closed center control valve, the pump remains in high pressure standby mode at the PC setting with zero flow until the function is actuated.

Typical operating curve



Simple closed center circuit



Once the closed center valve is opened, the PC control senses the immediate drop in system pressure and increases pump flow by increasing the swashplate angle.

The pump continues to increase flow until system pressure reaches the PC setting.

If system pressure exceeds the PC setting, the PC control reduces the swashplate angle to maintain system pressure by reducing flow. The PC control continues to monitor system pressure and changes swashplate angle to match the output flow with the work function pressure requirements.

If the demand for flow exceeds the capacity of the pump, the PC control directs the pump to maximum displacement. In this condition, actual system pressure depends on the actuator load.

For additional system protection, install a relief valve in the pump outlet line.

# Pressure compensated system characteristics

- Constant pressure and variable flow
- · High pressure standby mode when flow is not needed
- System flow adjusts to meet system requirements
- Single pump can provide flow to multiple work functions
- Quick response to system flow and pressure requirements

# Typical applications for pressure compensated systems

- Constant force cylinders (bailers, compactors, refuse trucks)
- On/off fan drives
- Drill rigs
- Sweepers
- Trenchers



### PVBS, main spools for pressure control

The spools are designed in such a way that the pump pressure is controlled by the spool travel. The main spool must be displaced until the pump pressure just exceeds the load pressure before the working function is applied. If the main spool is held in this position, the pump pressure will remain constant – even if the load pressure changes – giving a stable system.

The use of pressure control spools, however, also means that:

- · the oil flow is load dependent
- the dead band is load dependent
- the pump pressure can exceed the load pressure by more than is usual
- the pressure drop across main spool varies (energy consumption)

Due to these factors it is recommended that pressure control spools are only used when it is known for certain that problems with stability will arise or already have arisen, and in applications where constant pressure is needed e.g. drill holding.

#### **Background**

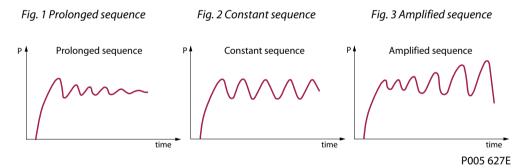
Instability in load sense control systems in certain applications with oscillations in the range of 1/2 - 2 Hz can cause severe instability problems while trying to control functions in an application.

Critical applications are usually related to functions with an important inertia torque and/or functions with secondarily fitted pressure controlled components e.g. over-center valves.

#### Examples.

- a slewing function
- main lifting/lowering function of a crane

The problem usually manifests itself in prolonged oscillation phenomena (Fig. 1), in a relatively constant sequence of oscillations (Fig. 2) or in the worst case in an amplified sequence of oscillations (Fig. 3).



To control the oscillation phenomena the "pressure control spool" was developed and is a patented system which can minimize most of the oscillation issues.

### **PVG 32 Principle**

The idea was to create a system operating independently of a constantly changing load pressure. Therefore, we changed the well-known LS principle (Fig. 4), so that compensated pump pressure is part of the LS system (Fig. 5) after the pressure compensator and before the metering range of the main spool. Upon actuation of the spool, it will be led via a fixed and a variable orifice.

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Fig. 4 Flow controlled spool

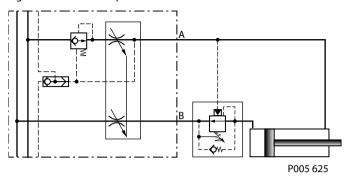
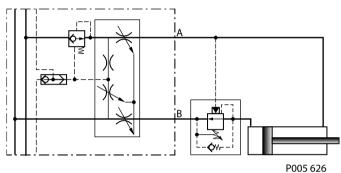


Fig. 5 Pressure controlled spool



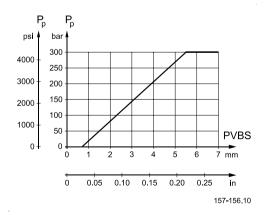
The opening area of the variable orifice is at maximum at initial actuation and 0 at full stroke of the spool and then the pressure created between the two orifices is led into the LS system in the usual way.

In this way the pump pressure is built up depending on the spool travel, e.g. the spool will then have to be stroked to a position that the pump pressure is higher than the actual load pressure to make the oil flow from  $P \rightarrow A/B$ .

When the load changes for a fixed spool position the flow to for the function will also change.

The valve section is now a load-dependent valve, but ensuring a constant pump pressure which is important in obtaining a stable function.

Pump pressure vs. spool travel curve



### **Pressure controlled spool application**

Pressure controlled spools should in principle only be used when you have stability issues. Typical applications on a crane:



- Lifting/lowering movement
- · Slewing movement with cylinders
- For the main lifting/lowering function on a crane it is recommended to fit a "half" pressure control
  spool. This means that the spool is designed with a normal flow control on the lifting port and
  pressure control connected to the port where the pilot signal to the over-center valve is acting. You
  will thus maintain a load-independent lifting movement and achieve a stable but load-depending
  lowering movement.
- As the load pressure on slewing movements is usually steady irrespective of the crane being loaded or not – it will be advantageous to use a "full" pressure control spool for A and B port.

In both cases we recommend the use of a basic valve, PVB, with pressure compensator. The pressure compensator will ensure the individual load-independency between the basic valves.

It is further recommended to use the LS pressure relief valves as not only will they ensure individual pressure limitation but also make it possible to adjust the maximum oil flow to the function.

It is not recommended to use shock valves as pressure limiting valves in connection with pressure control spools.

#### Sizing

The size of "half" (e.g: P - A = flow control P - B pressure control) pressure control spools is determined on basis of max. flow demand on the lifting port. If e.g. a max. pressure compensated flow of 65 l/min for the lifting movement, you choose a 65 L/min spool (size D). The metering characteristic has then a given size. As it is often requested to limit the use of the crane boom for downward push/force mode and the LS pressure limitation can be used. It will appear from the characteristics enclosed what effect a pressure limitation,  $P_{LS}$  will have on max. flow on the lowering port.

The size for a "full" pressure control spool is determined on basis of known load pressure, P<sub>LS</sub> max, and requested max. flow.

It will appear from the characteristics enclosed that if the load PLS is low and the pump pressure,  $P_p$ , is high as a result of max. stroked spool you will get a large flow.

If  $P_{LS}$  is approaching PLS max. the flow will be reduced and the dead band increased. Max. oil flow can be reduced by approx. 50% without limiting max. pressure.

The reduction is made by limiting the spool travel from 7 mm to 5.5 mm.

#### Limitation

If a pressure controlled spool is chosen for stability reasons consideration should be made to features related to the pressure control principle.

Deadband will change according to the load conditions and the valve section will become load-dependent and that the pump pressure may exceed the load pressure.

With all of the above in mind, a "pressure controlled spool" will minimize oscillation and obtain a stable function that can be controlled smooth and precise.



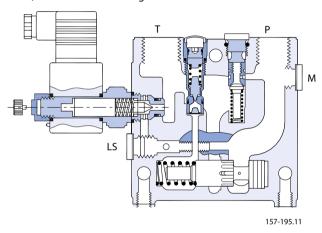
# PVPX, electrical LS unloading valve

PVPX is a solenoid LS unloading valve. PVPX is fitted into the pump side module enabling a connection to be made between the LS and the tank lines. Thus the LS signal can be relieved to tank by means of an electric signal.

For a PVP pump side module in open center version the relief to tank of the LS signal means that the pressure in the system is reduced to the sum of the tank port pressure plus the neutral flow pressure for the pump side module.

For a PVP pump side module in closed center version the relief to tank of the LS signal means that the pressure is reduced to the sum of the tank port pressure for the pump side module plus the stand-by pressure of the pump.

### PVPX, electrical LS unloading valve





#### **PVG 32 Technical Data**

### Operating parameters

Maximum rated pressure	A/B, P-port continuous <sup>1)</sup>	350 bar [5076 psi]	
	P-port intermittent <sup>2)</sup>	400 bar [5800 psi]	
	A/B-port intermittent <sup>2)</sup>	420 bar [6090 psi]	
	T-port static/dynamic	25/40 bar [363/580 psi]	
Oil rated flow	P-port <sup>3)4)</sup>	140/230 l/min [37/61 US gal/min]	
	Port A/B, with press. comp. <sup>5)</sup>	100 l/min [26.4 US gal/min]	
	Port A/B, without press. comp.	120 l/min [33 US gal/min]	
Max. internal leakage at 100 bar	$A/B \rightarrow T$ without shock valve	20 cm <sup>3</sup> /min [1.85 in <sup>3</sup> /min]	
[1450 psi]	$A/B \rightarrow T$ with shock valve	25 cm <sup>3</sup> /min [2.15 in <sup>3</sup> /min]	
Oil temperature	Recommended	30 to 60°C [86 to 140°F]	
	Minimum	-30°C [-22°F]	
	Maximum	90° [194°F]	
Ambient temperature	Recommended	-30 to 60°C [-22 to 140°F]	
Oil viscosity	Operating range	12 to 75 mm <sup>2</sup> /s [65 to 347 SUS]	
	Minimum	4 mm <sup>2</sup> /s [39 SUS]	
	Maximum	460 mm <sup>2</sup> /s [2128 SUS]	
Oil cleanliness	Minimum	23/19/16 (according to ISO 4406)	

<sup>1)</sup> With PVSI end plate; with PVS end plate max. 300 bar [4351 psi].

## Spool data

	Standard	± 7 mm [± 0.28 in]	
Spool travel	Proportional range	± 4.8 mm [± 0.19 in]	
	Float position	± 8 mm [± 0.32 in]	
Dead band, flow control spools	Standard	± 1.5 mm [± 0.06 in]	
Dead balld, flow control spools	Linear characteristic	± 0.8 mm [± 0.03 in]	

On standard PVB 32 modules, using main spools with closed neutral position, there will be a pressure build up on the A and B port, when main spool is in neutral, and high P pressure. The pressure build up equals to 0,5x P-pressure can be expected.

If this is causing problems on applications, please contact PVG sales team.

### Rated pressure

Product	P-port max. continuous pressure
PVG 32; PVG 120/32; PVG 100/32 with PVS	300 bar [4351 psi]
PVG 32; PVG 120/32; PVG 100/32 with PVSI	350 bar [5076 psi]
PVG 32 with PVBZ	250 bar [3626 psi]
PVG 32 with HIC steel	350 bar [5076 psi]
PVG 32 with HIC aluminum	210 bar [3046 psi]

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<sup>&</sup>lt;sup>2)</sup> Intermittent pressure at max. 250,000 cycles of full PVG life time cycles, with PVSI end plate. The maximum intermittent pressure at max. 250,000 cycles stresses the need to confirm application duty cycle before proceeding with specification. For further information contact Danfoss Product Application Engineering.

<sup>&</sup>lt;sup>3)</sup> In open circuit systems with short P-hoses/tubes, attention should be paid to pressure peaks at flows >100 l/min [26.4 US gal/min].

<sup>&</sup>lt;sup>4)</sup> For a system with mid inlet PVPVM.

<sup>&</sup>lt;sup>5)</sup> In open circuit systems with short P-hoses/tubes,For 130 l/min flow contact Danfoss Product Application Engineering.



## **PVG 32 Technical Data**

# **PVM operating force**

# Operating torque for PVM

Spool displacement		Operating torque N·m [lb·in]			
	PVM + PVMD, PVE	PVM + PVH	PVM + PVMR	PVM+PVMF	
From neutral position	2.2 ±0.2 [19.5 ±1.8]	2.5 ±0.2 [22.1 ±1.8]	17 [3.8]	22 [5.0]	
Max. spool travel	2.8 ±0.2 [24.8 ±1.8]	6.9 ±0.2 [61.0 ±1.8]	-	-	
Into float position	-	_	-	60 [13.5]	
Away from float position	-	-	_	28 [6.3]	
From any other position	-	-	8.5 [73.3]	_	

# PVH, hydraulic actuation

## PVH, Hydraulic Actuation

PVH data	Pressure, bar [psi]
Control range	5 to 15 [75 to 220]
Maximum pilot pressure, static	30 [435]
Maximum pressure on port T (It is recommended that the tank connection from the hydraulic remote control unit PVRH is taken directly to tank.)	10 [145]

## **PVE actuation literature reference**

For the information about the available electrohydraulic actuator variants, please see the following documents overview.

# PVE actuation literature reference

Title	Туре	Lit. Number
Electrohydraulic Actuators, PVEO/M/A/H/S Series 7	Technical Information	BC00000378
Electrohydraulic Actuators, PVE Series 4 and PVHC	Technical Information	BC0000050
Electrohydraulic Actuators, PVED-CC Series 5 ISObus	Technical Information	BC00000361
Electrohydraulic Actuators, PVED-CC Series 5 CANopen	Technical Information	BC00000354
Electrohydraulic Actuators, PVED-CC Series 4	Technical Information	BC00000107
Electrohydraulic Actuators, PVED-CX Series 4	Technical Information	BC0000068

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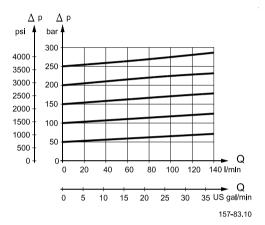
The measured characteristics in this catalog are typical measured values.

During measuring a mineral based hydraulic fluid with a viscosity of 21 mm $^2$ /s [102 SUS] at a temperature of 50 °C [122 °F] was used.

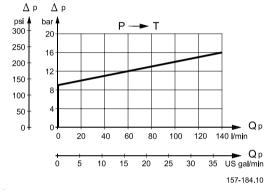
## **PVP pressure relief valve characteristics**

The pressure relief valve is set at an fluid flow of 15 l/min [4 US gal/min]. Pressure setting range for PVP with PVSI end plate is 30 to 350 bar [435 to 5075 psi] and for PVP with PVS end plate is 30 to 300 bar [435 to 4351 psi].

## Pressure relief valve characteristic



### Neutral by-pass pressure drop (open center)

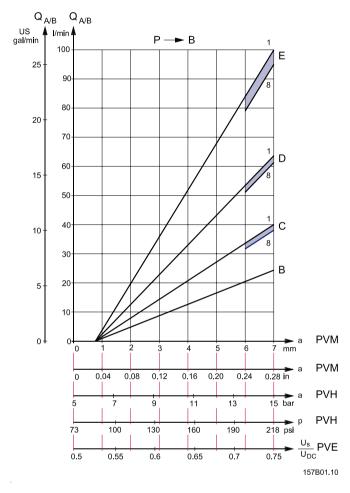




# PVB, basic modules oil flow characteristics

The fluid flow for the individual spool depends on type of basic module (with/without compensation) and type of pump (fixed or variable displacement).

Linear fluid flow depending on spool type



 $U_S$  = Signal voltage;  $U_{DC}$  = Supply voltage; 1 = First PVB after PVP; 8 = Eighth PVB after PVP



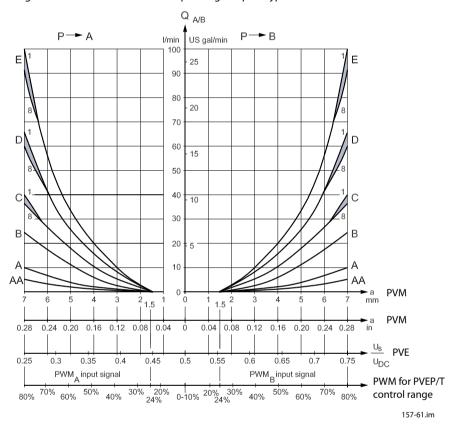
# Pressure-compensated PVB, open center PVP

The fluid flow is dependent on the supplied pump fluid flow.

The characteristics are plotted for a pump oil flow,  $Q_P$ , corresponding to the rated maximum spool oil flow,  $Q_N$ . Increasing the pump oil flow to  $1.4 \times Q_N$  will give the same oil flow on the eighth as on the first basic module.

The letters AA, A, B, C, D, E denote spool types. The characteristic below is shown for spool travel in both directions. All other characteristics are shown for spool travel in one direction only.

Progressive oil flow characteristic depending on spool type



 $U_S$  = Signal voltage;  $U_{DC}$  = Supply voltage; 1 = First PVB after PVP; 8 = Eighth PVB after PVP

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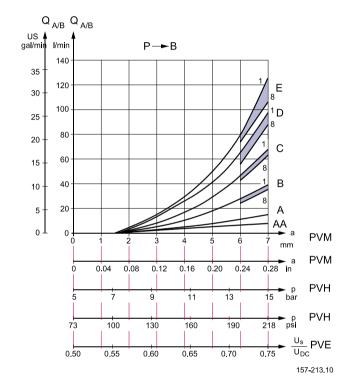


# PVB without pressure compensation, open center PVP

The spool flow  $Q_{A/B}$  is dependent on the supplied pump fluid flow,  $Q_P$ .

The characteristics apply to supply oil flow of 130 l/min [34.3 US gal/min] with the actuation of one basic module and the supply flow level. If several basic modules are activated at the same time, the characteristic depends on the load pressure of the actuated basic modules. The pressure drop of any oil flowing back to tank  $(Q_P - Q_{A/B})$  is read on the curve for neutral flow pressure in PVP.

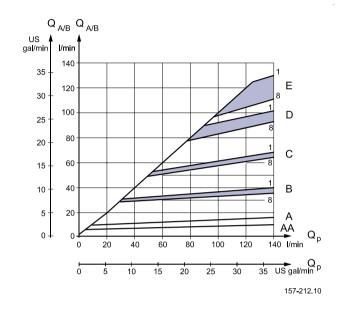
Fluid flow as a function of spool travel



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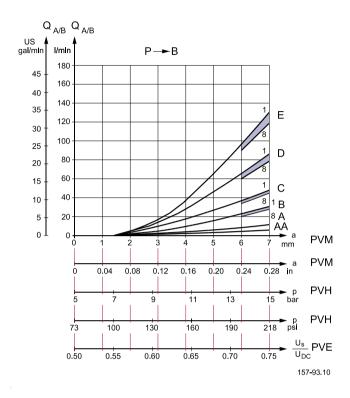


# Characteristic for fully displaced flow control spools

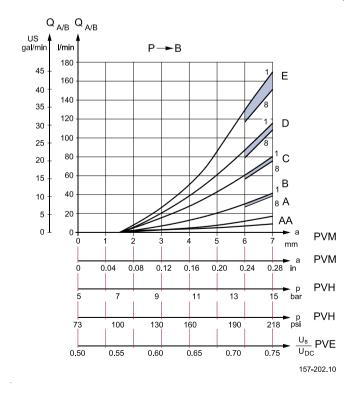




# PVB without pressure compensation, closed center PVP



Set pressure difference between pump pressure and LS signal = 10 bar [145 psi].



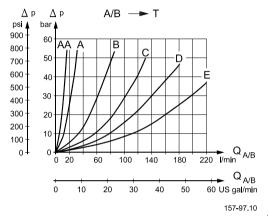
Set pressure difference between pump pressure and LS signal = 20 bar [290 psi].



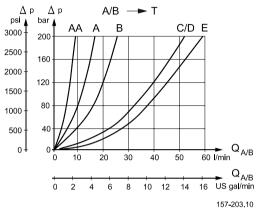
The oil flow is dependent on the pressure difference between the pump pressure and the LS signal. Normally the pressure difference is set at the LS pump regulator. Also take into consideration pressure drop from the pump to the PVG valve group. e.g. long pipeline.

### Oil flow characteristics for PVB at

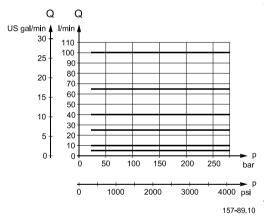
@ pressure drop at max. main spool travel



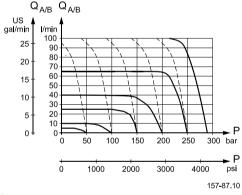
@ pressure drop for open spool in neutral position



Load-independent, pressure-compensated



LS pressure limiting, pressure-compensated PVB



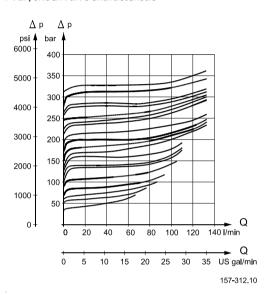


# PVLP, shock and PVLA, suction valves

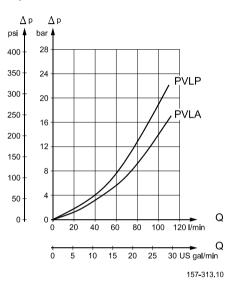
PVLP is set at an oil flow of 10 l/min [2.6 US gal/min]. The shock valve PVLP is designed to absorb shock effects. Consequently, it should not be used as a pressure relief valve.

If the working function requires the use of a pressure relief valve, a PVB basic module with built-in  $LS_{A/B}$  pressure limiting valve should be used.

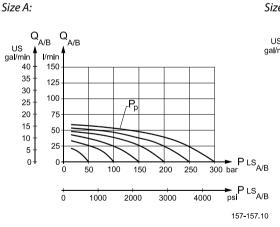
PVLP, shock valve characteristic

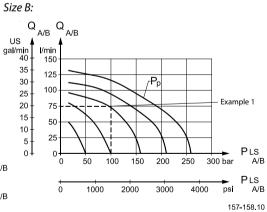


PVLA, suction valve characteristic

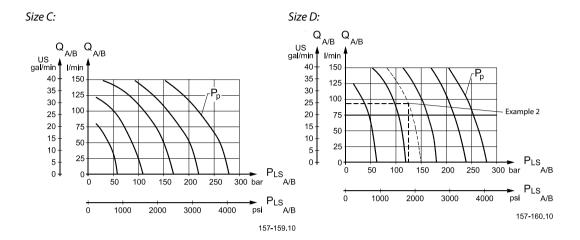


# Pressure control spool flow characteristics

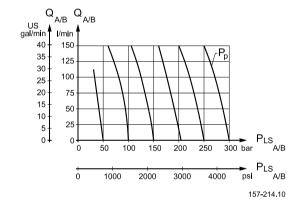








Size E:

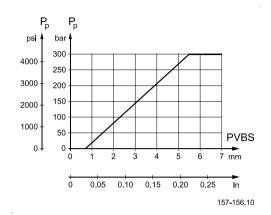


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## Pressure build-up for pressure controlled spools

Max. oil flow can be reduced by about 50% without limitation of maximum pressure by limiting the main spool travel from 7 mm [0.28 in] to 5.5 mm [0.22 in].



### Examples of how to use the characteristics for pressure control spools

#### **Example 1: Determining the oil flow**

#### Given:

- Spool type B
- Pressure setting P<sub>P</sub>: 160 bar [2320 psi]
- Load pressure, LS<sub>A/B</sub>: 100 bar [1450 psi] *Result:*

Oil flow = 75 l/min [19.8 US gal/min]

#### **Example 2: Determining the spool size**

#### Given

- Max. oil flow, Q<sub>A/B</sub>: 90 l/min [23.8 US gal/min]
- Pressure setting P<sub>P</sub>: 150 bar [2175 psi]
- Load pressure, P<sub>LSA</sub>: 125 bar [1810 psi] *Result*: D spool

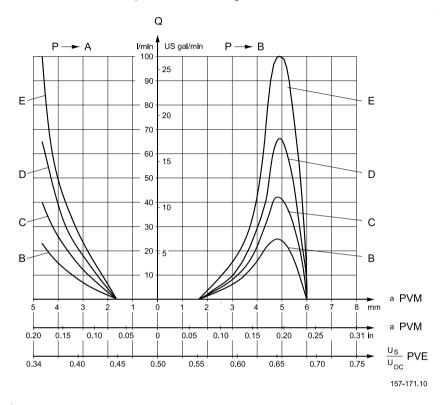
(see *Pressure CS flow characteristics*, size D)

Normally a smaller spool can be chosen with pressure control. It is our experience that the spool can be one size smaller than with normal flow control.



## **Characteristics for float position main spools**

Characteristic of oil flow, spool travel and voltage

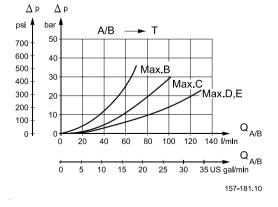


- 4.8 mm [0.19 in] spool displacement in direction A gives maximum oil flow to port A
- 4.8 mm [0.19 in] spool displacement in direction B gives maximum oil flow to port B
- 8 mm [0.32 in] spool displacement in direction B gives completely open float position A/B  $\rightarrow$  T.

The spools have 4.8 mm spool travel in direction A and 8 mm travel in direction B:

For more information regarding electrical actuation of float spools please see *PVE series 4 Technical Information*, **520L0553**.

Pressure drop  $A/B \rightarrow T$  at maximum spool travel within the proportional range (4.8 mm) [0.19 in]

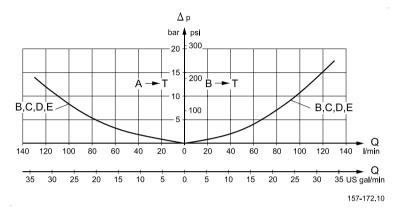


Spools D and E have the same opening area for forward flow and return flow.



Spool E can give 100 l/min [26.4 US gal/min] pressure compensated oil flow due to a higher pressure drop across spool E. This occurs during spool actuation only.

Pressure drop A/B  $\rightarrow$  T in float position

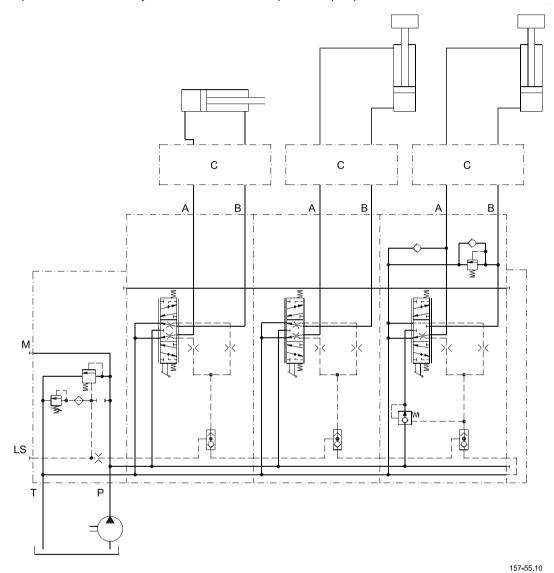




# **Hydraulic systems**

# Manually actuated PVG 32 – fixed displ. pump

Example schematic of manually actuated PVG 32 – fixed displacement pump



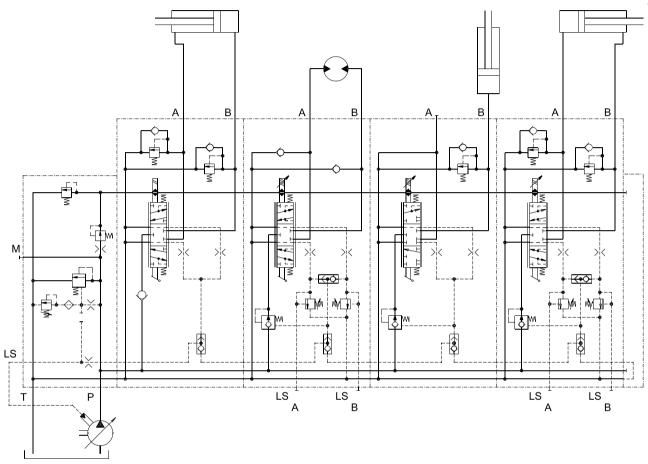
137-33.10



# **Hydraulic systems**

# Electrically actuated PVG 32 – variable displ. pump

Example schematic of electrically actuated PVG 32 – variable displacement pump (electrical actuator, shock valves, relief valve)



157-56.10



### **Fluid Specifications**

#### Oil

The main duty of the oil in a hydraulic system is to transfer energy; but it must also lubricate the moving parts in hydraulic components, protect them against corrosion, and transport dirt particles and heat out of the system. It is therefore important to choose the correct oil with the correct additives. This gives normal operation and long working life.

#### Mineral oil

For systems with this valve Danfoss recommends the use of mineral-based hydraulic oil containing additives: Type HLP (DIN 51524) or HM (ISO 6743/4).

#### Non-flammable fluids

Phosphate-esters (HFDR fluids) can be used without special precautions. However, dynamic seals must be replaced with FPM (Viton) seals. Please contact the Danfoss Sales Organization if the PVG 32 valve is to be used with phosphate-esters.

The following fluids should only be used according to agreement with the Danfoss Sales Organization for:

- Water-glycol mixtures (HFC fluids)
- · Water-oil emulsions (HFB fluids)
- · Oil-water emulsions (HFAE fluids)

#### **Filtration**

Effective filtration is the most important precondition in ensuring that a hydraulic system performs reliably and has a long working life. Filter manufacturers issue instructions and recommendations. It is advisable to follow these.

#### **System filters**

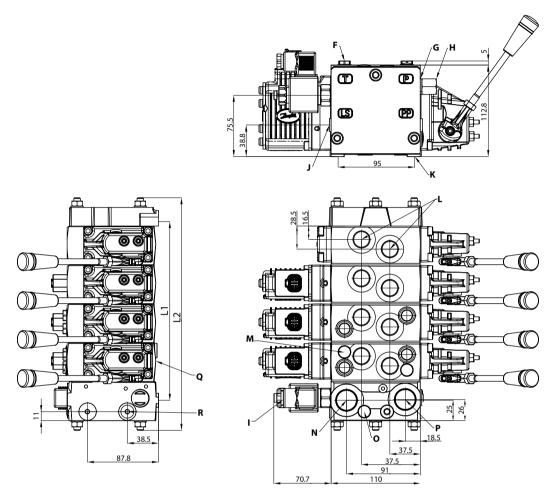
Where demands on safety and reliability are very high a pressure filter with bypass and indicator is recommended. Experience shows that a 10  $\mu$ m nominal filter (or finer) or a 20  $\mu$ m absolute filter (or finer) is suitable. It is our experience that a return filter is adequate in a purely mechanically operated valve system. The fineness of a pressure filter must be selected as described by the filter manufacturer so that a particle level of 23/19/16 is not exceeded. The filter must be fitted with pressure gauge or dirt indicator to make it possible to check the condition of the filter. In systems with differential cylinders or accumulators the return filter must be sized to suit the max. return oil flow. Pressure filters must be fitted to suit max. pump oil flow.

#### **Internal filters**

The filters built into PVG 32 are not intended to filter the system but to protect important components against large particles. Such particles can appear in the system as a result of pump damage, hose fracture, use of quick-couplings, filter damage, starting up, contamination, etc. The filter in the electrical actuator PVE protecting the solenoid valves has a mesh of 150  $\mu$ m. Bursting pressure drop for internal filters is 25 bar [360 psi].



### **Dimensions and ports description**



- F Shock and suction valve, PVLP: G¼, 12 mm [½–20, 0.47 in] deep
- **G** Pressure gauge connection: G¼, 12 mm [½–20, 0.47 in] deep
- H Plug for external pilot oil supply, PVPC: G½, 12 mm [½–20, 0.47 in] deep
- I Electrical LS unloading valve, PVPX
- **J** LS connection:  $G\frac{1}{4}$ , 12 mm  $\left[\frac{1}{2}-20; 0.47 \text{ in or } \frac{9}{16}-18, 0.5 \text{ in}\right]$  deep
- **K** Fixing holes: M8 × min. 10 mm  $[^{5}/_{16}$ –18; 0.39 in] deep
- **L** Port A and B:  $G\frac{1}{2}$ , 14 mm  $\left[\frac{7}{8}-14; 0.65 \text{ in}\right]$  deep
- **M** LS pressure limiting valve
- **N** Tank connection;  $G^{3/4}$ , 16 mm [1  $^{1}/_{16}$ –12; 0.75 in] deep
- O Pressure relief valve
- **P** Pump connection;  $G\frac{1}{2}$ , 14 mm [ $^{7}$ / $_{8}$ -14; 0.65 in] or  $G\frac{3}{4}$ , 16 mm [ $^{1}$ 1/ $_{16}$ -12; 0.75 in] deep
- **Q** LS<sub>A</sub> and LS<sub>B</sub> connections; G¼, 12 mm  $[^9/_{16}$ –18, 0.5 in] deep
- R Pilot pressure connection G, Pp

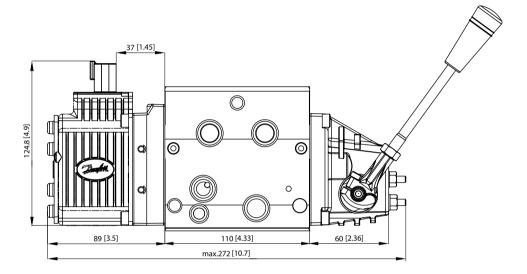


# **Dimensions and ports description**

## L1 and L2 dimensions

PVB		1	2	3	4	5	6	7	8	9	10	11	12
L1	mm	82	130	178	226	274	322	370	418	466	514	562	610
	[in]	[3.23]	[5.12]	[7.01]	[8.90]	[10.79]	[12.68]	[14.57]	[16.46]	[18.35]	[20.24]	[22]	[24]
L2	mm	140	189	238	287	336	385	434	483	527	576	622	670
	[in]	[5.51]	[7.44]	[9.37]	[11.30]	[13.23]	[15.16]	[17.09]	[19.02]	[20.95]	[22.87]	[24.5]	[26.4]

PVG 32 section dimensions

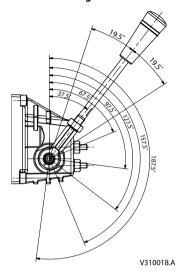




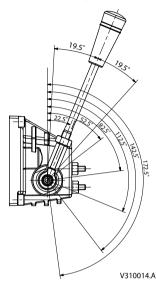
### **Dimensions and ports description**

## PVM, control lever positions

#### Base with an angle of 37.5°



#### Base with an angle of 22.5°



No control lever position	2×6
Control lever range	±19.5°
Proportional control lever range	±13.4°
Control lever range – float position	22.3°

The angle of the handle is determined by which side of the handle that is mount towards the base.

- If a 22.5° angle is needed the "dot" on the handle is not visible
- If 37.5° is needed the dot should be visible

#### **Surface treatment**

The PVG valve has as standard, an untreated surface. In certain applications, depend on different factors, such as: salty environment, large temperature changes, high humidity, rust can develope on the surface. This will not affect the performance of the PVG valve group. To prevent/reduce rust development, Danfoss recommend the PVG valve group to be painted. Rust on the surface is not seen as a valid complaint issue, neither on painted or unpainted PVG valve groups.



# **PVP schematics and code numbers**

Symbol	Description		Code number
TM	Open center pump side module for pumps with fixed displacement.	P = G ½ T = G ¾	157B5000
	For purely mechanically actuated valve groups	P = 7/8-14 T = 1 1/16-12	157B5200
P   \$		P, T = G 3/4	157B5100
157-24.11		P, T = 1 1/16–12	157B5300
TM	Closed center pump side module for pumps with vaiable displacement.	P = G ½ T = G ¾	157B5001
		P = 7/8-14 T = 1 1/16-12	157B5201
P	For purely mechanically actuated valve	P, T = G 3/4	157B5101
157-23.11	groups.	P, T = 1 1/16–12	157B5301
TF	Open center pump side module for pumps with fixed displacement.	P = G ½ T = G ¾	157B5010
	With pilot oil supply for electrically actuated valves.	P = 7/8-14 T = 1 1/16-12	157B5210
╒┊┃╶┿╌╌╌╌╅╌╩┸┸┸┆		P, T = G 3/4	157B5110
157-22.10		P, T = 1 1/16–12	157B5310
LS M	Closed center pump side module for pumps with variable displacement.	$P = G \frac{1}{2}$ $T = G \frac{3}{4}$	157B5011
	With pilot oil supply for electrically actuated valves.	P = 7/8-14 T = 1 1/16-12	157B5211
		P, T = G 3/4	157B5111
157-21.10		P, T = 1 1/16–12	157B5311
_LS _M	Open center pump side module for pumps with fixed displacement.	P = G ½ T = G ¾	157B5012
√ 1 × 1 × 1	With pilot oil supply for electrically actuated valves Connection for electrical LS unloading valve,	P = 7/8-14 T = 1 1/16-12	157B5212
	PVPX (not incl.)	P, T = G 3/4	157B5112
		P, T = 1 1/16–12	157B5312
. 157-153.11			



Symbol	Description		Code number
LS M	Closed center pump side module for pumps with variable displacement	P = G ½ T = G ¾	157B5013
	Connection for electrical LS unloading valve, PVPX (not incl.)	P = 7/8-14 T = 1 1/16-12	157B5213
		P, T = G 3/4	157B5113
Pi x x - J - 7		P, T = 1 1/16–12	157B5313
157-154.10			

#### Connections:

 $P = G \frac{1}{2}$  in; 14 mm deep or G  $\frac{3}{4}$  in; 16 mm deep / LS, M = G  $\frac{1}{4}$  in; 12 mm deep / T = G  $\frac{3}{4}$  in; 16 mm deep.  $P = \frac{7}{8} - 14$ ; 0.65 in deep or 1  $\frac{1}{16} - 12$ ; 0.75 in deep / LS, M =  $\frac{1}{2} - 20$ ; 0.47 in deep / T = 1  $\frac{1}{16} - 12$ ; 0.75 in deep.

## PVP, pump side modules

Symbol	Description		Code number
NO 11 20 M M T T T T T T T T T T T T T T T T T	Open center pump side module for pumps with fixed displacement. For mechanical actuated valves. Connection for LS unloading valve, PVPX (not incl)	P, T = G ¾	157B5102
157-295.10	Closed center pump side module for pumps with vaiable displacement. For mechanical actuated valves. Connection for LS unloading valve, PVPX (not incl)	P, T = G ¾	157B5103
157-243.11	Open center pump side module for pumps with fixed displacement. With pilot oil supply for electrical actuation and connection for pilot oil pressure Incl. check valve	P, T = G ¾  P, T = 1 1/16–12 LS connection = 9/16–18	157B5180 157B5380

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## PVP, pump side modules (continued)

Symbol	Description		Code number
157-523.10	Closed center pump side module for pumps with variable displacement. With pilot oil supply for electrical actuation and connection for pilot oil pressure Incl. check valve	P, T = G <sup>3</sup> / <sub>4</sub> P, T = 1 1/16–12 LS connection = 9/16–18	157B5181 157B5381
M M M M M M M M M M M M M M M M M M M	Open center pump side module for pumps with fixed displacement. With pilot oil supply for hydraulic actuation and connection for pilot oil pressure	P, T = G <sup>3</sup> / <sub>4</sub> P, T = 1 1/16–12 LS connection = 9/16–18	157B5190 157B5390
157-245.10	Closed center pump side module pumps with variable displacement With pilot oil supply for hydraulic actuation and connection for pilot oil pressure	P, T = G <sup>3</sup> / <sub>4</sub> P, T = 1 1/16–12 LS connection = 9/16–18	157B5191 157B5391

## Connections:

P, T = G  $\frac{3}{4}$  in; 16 mm deep / LS, M = G  $\frac{1}{4}$  in; 12 mm deep

P, T = 1 1/16-12; 0.75 in deep / LS, M =  $\frac{1}{2}-20$ ; 0.47 in deep.

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# PVB, basic module code numbers

PVB without adjustable  $LS_{A/B}$  pressure limiting valves

Symbol	Description		Code number	
			No facilities for shock valves A/B	Facilities for shock valves A/B
; <del>                                     </del>	Without load drop check valve and pressure compensator.	G ½ 14 mm deep	157B6000	157B6030
1 0 2 M	Can be used where load holding valves prevent oil from flowing back through channel P. A	7/8–14 0.65 in deep	157B6400	157B6430
157-19	Load drop check valve.	G ½ 14 mm deep	157B6100	157B6130
157-20	В	7/8–14 0.65 in deep	157B6500	57B6530
·	Load drop check valve. LS <sub>A/B</sub> shuttle valve.	G ½ 14 mm deep	_	157B6136
1 0 2 M	To be used with float position spools.  A	7/8–14 0.65 in deep		157B6536
157-196	10			

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PVB without adjustable  $LS_{A/B}$  pressure limiting valves (continued)

Symbol	Description		Code number	
			No facilities for shock valves A/B	Facilities for shock valves A/B
[H]	Non-damped compensator valve	G ½ 14 mm deep	157B6200	157B6230
M 1 0 2 M A A A A A A A A A A A A A A A A A A		7/8–14 0.65 in deep	157B6600	157B6630
	Without compensator valve LS <sub>A/B</sub> shuttle valve	G ½ 14 mm deep	_	11071832
M 1 0 2 M A A A A A A A A A A A A A A A A A A		7/8–14 0.65 in deep	_	_
<u> </u>	With damped compensator valve	G ½ 14 mm deep	157B6206	157B6236
1 0 2 M A		7/8–14 0.65 in deep	11036629	11036630
157-16.10				



PVB with adjustable  $LS_{A/B}$  pressure limiting valves

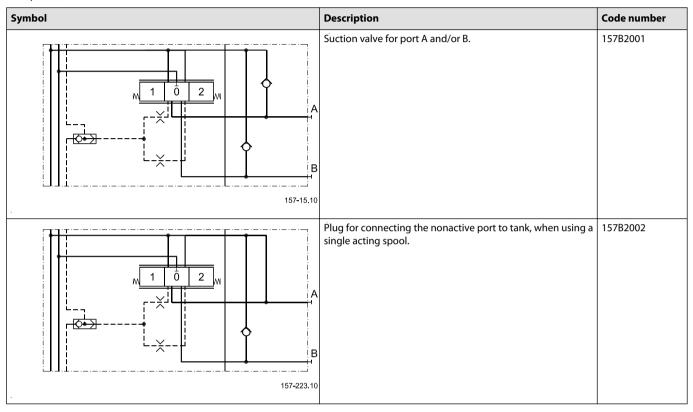
Symbol	Description		Code n	umber
			No facilities for shock valves A/B	Facilities for shock valves A/B
	With non-damped compensator valve Adjustable LS <sub>A/B</sub> pressure limiting	G ½ 14 mm deep	157B6203	157B6233
LS <sub>A</sub> 1 0 2 M A A 157-198.10	valves External LS connection port A/B. Also used for float position spools	7/8–14 0.65 in deep	157B6603	157B6633
[ ] [ ] [ ] [ ] [ ] [ ] [ ] [ ] [ ] [ ]	Damped compensator valve Adjustable LS <sub>A/B</sub> pressure limiting	G ½ 14 mm deep	157B6208	157B6238
LS <sub>A</sub> 1 0 2 <sub>M</sub> A	valves External LS connection port A/B	7/8–14 0.65 in deep	-	11036631
157-17.10				

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# PVLA, suction valve (fitted in PVB)

## PVLA, suction valve



#### PVLP, shock and suction valve (fitted in PVB)

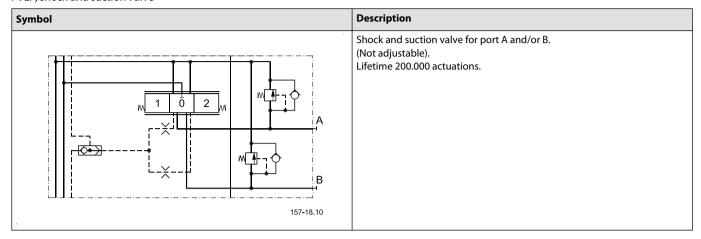
## PVLP, shock/ and anti-cavitation valves

Code no. 157B		2032	2050	2063	2080	2100	2125	2140	2150	2160	2175	2190
Settings	bar	32	50	63	80	100	125	140	150	160	175	190
	[psi]	[460]	[725]	[914]	[1160]	[1450]	[1813]	[2031]	[2175]	[2320]	[2538]	[2755]

Code no. 157B		2210	2230	2240	2250	2265	2280	2300	2320	2350	2380	2400
Settings	bar	210	230	240	250	265	280	300	320	350	380	400
	[psi]	[3045]	[3335]	[3480]	[3625]	[3845]	[4061]	[4351]	[4641]	[5075]	[5511]	[5801]



## PVLP, shock and suction valve



# PVM, PVMD, PVMR, and PVMF code numbers

#### PVM, mechanical actuation

Symbol	Description	Code number with stop screws w/o stop screws		
	PVM, Standard, spring centered Individual oil flow adjustment to ports A and B	157B3171	157B3191	
1 0 2 M 157-10.10	Without actuation lever and base. Shaft for mounting of actuation lever	157B3173	157B3193	
	PVM, as standard, witout actuation lever. With base for mounting of actuation lever	157B3174	157B3194	
	PVM, Standard, spring. Individual oil flow adjustment to ports A and B. (Anodized)	157B3184	-	

#### PVMD, cover for mechanical actuation

Symbol	Description	Material	Code No.	Anodized
_	PVMD, Cover for purely mechanically operated valve	aluminium	157B0001	no
		aluminium	157B0009	yes
		cast iron	157B0021	no

#### PVMR, friction detent

Symbol	Description	Material	Code number	Anodized
	PVMR, Friction detent	aluminium	157B0004	no
1 0 2		aluminium	157B0012	yes
		cast iron	157B0024	-
157-210.10				

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#### PVMF, mechanical float position

Symbol	Description	Material	Code number	Anodized
M 1 0 2 F W	PVMF, Mechanical float position lock	aluminium	157B0005	no
157-208.10				
M F 1 0 2 M				

## PVH, hydraulic actuation code numbers

## PVH, hydraulic actuation

Symbol	Description	Material	Code number	Anodized
1 0 2	PVH, Cover for Hydraulic actuation	aluminum	157B0007	no
157-199.11	PVH 9/16-18 UNF	aluminum	157B0010	yes
		cast iron	157B0014	no
	PVH, Cover for Hydraulic actuation	aluminum	157B0008	no
	PVH G1/4	aluminum	157B0011	yes
		cast iron	157B0016	no

### **PVE actuation literature reference**

For the information about the available electrohydraulic actuator variants, please see the following documents overview.

#### PVE actuation literature reference

Title	Туре	Lit. Number
Electrohydraulic Actuators, PVEO/M/A/H/S Series 7	Technical Information	BC00000378
Electrohydraulic Actuators, PVE Series 4 and PVHC	Technical Information	BC0000050
Electrohydraulic Actuators, PVED-CC Series 5 ISObus	Technical Information	BC00000361
Electrohydraulic Actuators, PVED-CC Series 5 CANopen	Technical Information	BC00000354
Electrohydraulic Actuators, PVED-CC Series 4	Technical Information	BC00000107
Electrohydraulic Actuators, PVED-CX Series 4	Technical Information	BC0000068

#### PVS, end plate

### PVS, end plate

Symbol	Description	Mounting threads	Code number		
[[1]	PVS, without active elements.	BSP	157B2000		
V310062.A	No connections	SAE	157B2020		
[1-1	PVS, without active elements.	G 1/8 10 mm deep	BSP	157B2011	
LX V310063.A	Max. intermittend LX pressure 250 bar [3625 psi]	3/8 in - 24; 0,39 in deep	SAE	157B2021	



## PVS, end plate (continued)

Symbol	Description	Description					
[[±]]	PVSI, without active elements		BSP	157B2014			
V310062.A	Without connections.	Without connections.					
<u> </u>	PVSI, without active elements	G 1/4 10 mm deep	BSP	157B2015			
LX V310063.A	LX connections.  Max. intermittend LX pressure:  350 bar [5075 psi]	1/2 in - 20; 0,47 in deep	SAE	157B2005			

For mounting threats please see the chapter *Dimensions*.

## PVAS, assembly kit code numbers

## PVAS, assembly kit code numbers

	Code numbers 157B												
No.	0	1	2	3	4	5	6	7	8	9	10	11	12
PVB's	8000	8001	8002	8003	8004	8005	8006	8007	8008	8009	8010	8061	8062
PVB + PVPVM	-	8021	8022	8023	8024	8025	8026	8027	8028	8029	8030	8081	8082
Weight kg [lb]	0.1 [0.2]	0.15 [0.3]	0.25 [0.6]	0.30 [0.7]	0.40 [0.9]	0.45 [1.0]	0.50 [1.1]	0.60 [1.3]	0.65 [1.4]	0.70 [1.6]	0.80 [1.7]	0.85 [1.8]	0.9 [2.0]

#### PVPX, electrical LS unloaded valve

## PVPX, electrical LS unloaded valve

Symbol	Description		Code number
	PVPX, Normally open:	12 V	157B4236
W[.]♥ □	LS pressure relieved with no signal to PVPX	24 V	157B4238
157-150.10			
	PVPX, Normally closed:	12 V	157B4246
M <mark>∳ ↓</mark> □	LS pressure relieved with signal to PVPX	24 V	157B4248
157-151.10			
	PVPX, Normally open with manual override:	12 V	157B4256
157-152.10	LS pressure relieved with no signal to PVPX Manual override DE-selects LS-pump	24 V	157B4258
-	Plug		157B5601

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# PVPC, plug for external pilot oil supply

PVPC, plug for external pilot oil supply

Symbol	Description		Code number
T M M	PVP, Plug without check valve for open	G 1/2, 12 mm deep	157B5400
157-191.10	or closed center	1/2 in - 20; 0.47 in deep	-
T, M	PVP, Plug with check valve for	G 1/2, 12 mm deep	157B5600
	open center	1/2 in - 20; 0.47 in deep	157B5700
157-192.10			

# **Standard FC spools**

PVB is	with LS	<sub>A/B</sub> shutt	tle valve				Code number 15	PVB is without LS <sub>A/B</sub> shuttle valve							
Press. o	compens	sated flo	w: I/min	[US gal/ı	min]		ISO symbol	Symbol	Press. o	ompens	sated flo	w I/min	[US gal/ı	min]	
F 130 [34.3]	E 100 [26.4]	D 65 [17.2]	C 40 [10.6]	B 25 [6.6]	A 10 [2.6]	AA 5 [1.3]			AA 5 [1.3]	A 10 [2.6]	B 25 [6.6]	C 40 [10.6]	D 65 [17.2]	E 100 [26. 4]	F 130 [34.3]
7026	7024	7023	7022	7021	7020	7025	B A  P T 157-02.11  B A  TPT  157-26.  4-way, 3-position Closed neutral pos		7005	7000	7001	7002	7003	700	7006
7126	7124	7123	7122	7121	7120	7125	B A  P T 157-03.11  B A  TPT  157-23  4-way, 3-position Throttled, open no		7105	7100	7101	7102	7103	710	7106



PVB is	with LS,	<sub>A/B</sub> shutt	le valve				Code number 157B	PVB is without LS <sub>A/B</sub> shuttle valve						
-	-	-	-	-	-	-	A  P T  157-04.10   A  T A  T T T  T P T  T P T  157-28.10  3-way, 3-position  Closed neutral position, $P \rightarrow A$	-	7200	7201	7202	7203	720 4	-
-	-	-	-	-	-	-	B  B  T  T  T  T  T  T  T  T  T  T  T  T	-	-	7301	7302	7303	730 4	-
-	7424	7423	7422	7421	-	-	B A  P T  157-06.10  B A  TP T  157-30.10  4-way, 3-position  Throttled, $A \rightarrow T$ in neutral position	-	-	7401	7402	7403	740	7406



PVB is	B is with LS <sub>A/B</sub> shuttle valve				Code number 157B	PVB is	without	t LS <sub>A/B</sub> sl	huttle v	alve				
-	7524	7523	7522	7521	-	-	B A  P T  157-07.10  BA  TPT  157-31.10  4-way, 3-position  Throttled, $B \rightarrow T$ in neutral position	-	-	7501	7502	7503	750 4	-
-	7624	7623	7622	7621	7620	-	BA  PT  157-139.10  BA  TPT  157-140.10  4-way, 4-position  Closed neutral position  Float $P \rightarrow B \rightarrow F$	-	-	-	-	-	-	-

# Standard FC spools, hydraulic actuation

PVB is	with LS,	<sub>A/B</sub> shutt	le valve			Code number 157B		PVB is	without	LS <sub>A/B</sub> sh	nuttle va	lve	
Press. o	compens	ated flov	v: l/min	[US gal/r	nin]	ISO symbol	Symbol	Press. c	ompens	ated flov	w: l/min [	[US gal/n	nin]
E 100 [26.4]	D 65 [17.2]	C 40 [10.6]	B 25 [6.6]	A 10 [2.6]	AA 5 [1.3]			AA 5 [1.3]	A 10 [2.6]	B 25 [6.6]	C 40 [10.6]	D 65 [17.2]	E 100 [26.4]
9024	9023	9022	9021	9020	9025	B A P T 157-02.11  4-way, 3-position closed neutral position	B A  TPT  157-117.10	9005	9000	9001	9002	9003	9004
9124	9123	9122	9121	9120	9125	B A  P T 157-03.11  4-way, 3-position  Throttled open neutral p	BA	9105	9100	9101	9102	9103	9104



# FC spools for mechanical float position, PVMF

PVB is	PVB is with LS <sub>A/B</sub> shuttle valve Press. compensated flow: I/min [US gal/min]						Code number 157E	3	PVB is	withou	t LS <sub>A/B</sub> s	shuttle v	valve		
Press.	compen	sated flo	w: I/mir	ı [US ga	l/min]		ISO symbol	Symbol	Press.	compen	sated flo	ow l/min	[US gal	/min]	
F 130 [34.3]	E 100 [26.4]	D 65 [17.2]	C 40 [10.6]	B 25 [6.6]	A 10 [2.6]	AA 5 [1.3]			AA 5 [1.3]	A 10 [2.6]	B 25 [6.6]	C 40 [10.6]	D 65 [17.2]	E 100 [26.4]	F 130 [34.3]
-	9824	9823	9822	9821	9820	9825	B A  P T  157-09.10  B A  T P T  157-454.10  4-way, 4 position  Closed neutral position $P \rightarrow A \rightarrow F$			-	-	-	-	-	-
-	9624	623	9622	9621	-	-	B A  P T  157-139.10  4-way, 4-position  Closed neutral posit  Float $P \rightarrow B \rightarrow F$	BA TPT 157-140.10	-	-	-	-	-	-	-

## FC spools for friction detent, PVMR

PVB is	with LS	<sub>A/B</sub> shutt	le valve	1		Code number 157B		PVE	is withou	t LS <sub>A/B</sub> s	huttle va	lve	
Press. o	compens	ated flov	w: l/min	[US gal/r	nin]	ISO symbol	Symbol	Pres	s. compen	sated flo	w: l/min	[US gal/r	nin]
E 100 [26.4]	D 65 [17.2]	C 40 [10.6]	B 25 [6.6]	A 10 [2.6]	AA 5 [1.3]			AA 5 [1.3]	A 10 [2.6]	B 25 [6.6]	C 40 [10.6]	D 65 [17.2]	E 100 [26.4]
9724	9723	9722	9721	9720	-	B A P T 157-02.11  4-way, 3-position closed neutral position	BA 	-	9700	9701	9702	9703	9704
9734	9733	9732	9731	9730	-	B A  P T 157-03.11  4-way, 3-position  Throttled open neutral po	B A TPT 157-118.10	-	9710	9711	9712	9713	9714

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# FC spools with linear flow characteristic

PVB is	with LS <sub>A</sub>	B shuttle	e valve			Code number 157B	PVB is	without	LS <sub>A/B</sub> sh	uttle valv	re	
Press. c	ompensa	ited flow	: l/min [U	S gal/mii	n]	ISO symbol Symbol	Press. c	ompensa	ated flow	: I/min [U	S gal/mii	n]
E 100 [26.4]	D 65 [17.2]	C 40 [10.6]	B 25 [6.6]	A 10 [2.6]	AA 5 [1.3]		AA 5 [1.3]	A 10 [2.6]	B 25 [6.6]	C 40 [10.6]	D 65 [17.2]	E 100 [26.4]
9774	9773	9772	9771	-	-	B A    B A	-	9750	9751	9752	9753	9754
9784	9783	9782	9781	-	-	B A  B A  T P T  157-27.10  4-way, 3-position Throttled, open neutral position	-	9760	9761	9762	9763	9764
-	-	-	-	-	-	B A  P T  157-06.10  B A  T T T T  T P T  157-30.10  4-way, 3-position Throttled, A $\rightarrow$ T in neutral position	-	-	-	-	-	9794
-	-	-	-	-	-	B A  B A  B A  B A  THE THE STATE OF THE ST	-	-	-	-	-	9804

# **Standard PC spools**

PVB is	with LS,	<sub>A/B</sub> shutt	le valve			Code number 157B		PVB is	PVB is without LS <sub>A/B</sub> shuttle valve					
Press. o	ress. compensated flow: I/min [US gal/min]				min]	ISO symbol	Symbol	Press.	Press. compensated flow: I/min [US gal/min]				nin]	
E	D	С	В	Α	AA			AA	Α	В	С	D	Е	
100	65	40	25	10	5			5	10	25	40	65	100	
[26.4]	4] [17.2] [10.6] [6.6] [2.6] [1.3]			[1.3]			[1.3]	[2.6]	[6.6]	[10.6]	[17.2]	[26.4]		



PVB is	with LS,	<sub>A/B</sub> shutt	le valve			Code number 157B	PVB is	without	: LS <sub>A/B</sub> sł	nuttle va	lve	
-	7033	7032	7031	7030	7035	B A  B A  P T  TPT  157-143.10  4-way, 3-position  Closed neutral position, $PC \rightarrow A$ and B	7015	7010	7011	7012	7013	
7134	7133	7132	7131	7130	7135	B A  B A  P T  TPT  157-146.10  157-128.10  4-way, 3-position Throttled, open neutral position, PC → A and B	7115	7110	7111	7112	7113	-
7064	7063	7062	7061	-	-	B A B A B A T T P T T P T T T T T T T T T T T T T	-	7040	7041	7042	7043	7044
7074	7073	7072	7071	-	-	B A  B A  P T  TPT  157-145.10  157-122.10  4-way, 3-position  Closed neutral position, $PC \rightarrow B$	-	7050	7051	7052	7053	7054
7164	7163	7162	7161	-	-	B A  P T  157-147.10  B A  TPT  157-130.10  4-way, 3-position  Throttled, open neutral position, PC $\rightarrow$ A	-	7150	7151	7152	7153	7154
7174	7173	7172	7171	-	-	B A B A P T TPT 157-148.10 157-132.10  4-way, 3-position Throttled, open neutral position, PC → B	-	7150	7151	7152	7153	7154

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PVB is	with LS,	<sub>A/B</sub> shutt	le valve			Code number 157B	PVB is	without	LS <sub>A/B</sub> sl	huttle va	lve	
-	7473	7472	7471	7470	-	B A B A TPT TPT 157-149.10  4-way, 3-position Throttled, $A \rightarrow T$ neutral position, $PC \rightarrow B$	-	-	-	7452	7453	-
-	7563	7562	-	-	-	$\begin{array}{c c} & B & A & & B & A \\ \hline P & T & & & TP & T \\ \hline 157-167.10 & & 157-188.10 \\ \hline 4-way, 3-position \\ Throttled, B \rightarrow T neutral position , PC \rightarrow A \\ \end{array}$	-	-	7541	7542	7543	-

# Standard PC spools, hydraulic actuation

PVB is	VB is with LS <sub>A/B</sub> shuttle valve ress. compensated flow: I/min [US gal/min]					Code number 157B		PVB is	without	: LS <sub>A/B</sub> sh	nuttle va	lve	
Press. c	ompens	ated flo	v: l/min	[US gal/r	min]	ISO symbol	Symbol	Press. o	ompens	ated flov	w: l/min [	[US gal/r	nin]
E 100 [26.4]	D 65 [17.2]	C 40 [10.6]	B 25 [6.6]	A 10 [2.6]	AA 5 [1.3]			AA 5 [1.3]	A 10 [2.6]	B 25 [6.6]	C 40 [10.6]	D 65 [17.2]	E 100 [26.4]
-	-	-	-	-	-	B A  P T  157-143.10  4-way, 3-position Closed neutral position,	B A  TPT  157-121.10  PC $\rightarrow$ A and B	9015	9010	9011	9012	-	-
-	-	-	-	-	-	B A  P T  157-144.10  4-way, 3-position Closed neutral position,	BA TPT 157-123.10 PC → A	-	-	-	9042	9043	9044
-	-	-	-	-	-	B A  P T  157-145.10  4-way, 3-position  Closed neutral position,	BA TPT 157-122.10 PC → B	-	-	-	9052	9053	9054



### **Order Specification**

#### **PVG 32 order specification**

The form can be obtained from the Danfoss Sales Organization. An order form for PVG 32 hydraulic valve is shown on the page *PVG 32 specification sheet* on page 66. Both the module selection chart on the previous pages and the order form are divided into fields 0, 1-14, 15, 16, 17, a, b, c, e, f.

Each module has its own field:

0 Inlet section including:

Pump side module PVP

• Plug for external pilot oil supply PVPC

· Electrical LS unloading valve PVPX

1-14 Work section including:

a: Mechanical actuator, PVM (or PVE when option mounted)

b: Shock/suction valve, PVLP and suction valve, PVLA

c:

Cover for mechanical actuation, PVMD

Cover for hydraulic actuation, PVH

• Electrical actuators, PVE (or PVM when option mounted)

e: Main spool PVBS

f: Basic module, PVB

15 End plate, PVS

**16 Painting** Assembly kit, PVAS

#### Please state:

- Required code numbers of all modules
- Required setting (P) for pump side module
- Required setting of LS<sub>A/B</sub> pressure limiting valves, see pressure setting guidance below.

#### Reordering

The space at the top right-hand corner of the form is for Danfoss to fill in. The code number for the whole of the specified valve group (PVG No.) is entered here.

In the event of a repeat order all you have to do is enter the number Danfoss has given on the initial confirmation of order.



### **Order Specification**

## **Pressure setting limits**

The maximum setting pressure for the pressure limiting valves  $LS_A$  or  $LS_B$  depends on the chosen pressure setting for shock valve PVLP. The maximum values recommended to avoid interaction can be read in the following table.

The figures in the table have been calculated according to the following expressions:

- PVLP  $\leq$ 150 bar: LS<sub>A/B</sub>  $\leq$  0.8  $\times$  P<sub>PVLP</sub>
- PVLP > 150 bar:  $P_{PVLP}$   $LS_{A/B} \ge 30$  bar.

Max. pressure setting of  $LS_A$  and  $LS_B$  valves relative to PVLP shock valve

Pressure for PVL	P	Max. for LS <sub>A/B</sub>		Min. for LS <sub>A/B</sub>	
bar	[psi]	bar	[psi]	bar	[psi]
32	[460]	-	-	30 bar]	[435 psi]
50	[725]	40	[580]	1	
63	[914]	50	[720]	]	
80	[1160]	64	[930]	]	
100	[1450]	80	[1160]	]	
125	[1813]	100	[1450]	1	
140	[2031]	112	[1625]	]	
150	[2175]	120	[1740]	1	
160	[2320]	130	[1885]	1	
175	[2838]	145	[2100]	1	
190	[2755]	160	[2320]	1	
210	[3045]	180	[2610]	1	
230	[3335]	200	[2900]	1	
240	[3480]	210	[3045]	1	
250	[3625]	220	[3190]	1	
265	[3843]	235	[3408]	1	
280	[4061]	250	[3625]	1	
300	[4351]	270	[3915]	1	
320	[4641]	290	[4205]	1	
350	5075	320	4641	1	
380	5511	350	5075	1	
400	5801	370	5366	1	



### **Order Specification**

## **PVG 32 specification sheet**

Danfoss	
Subsidiary / Dealer	

PVG 32 Specification Sheet

Subsidiary / Dealer	PVG No.	
,		
Customer	Customer No.	
Application	Revision No.	

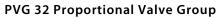
	Function	A-port							B-port
0	Inlet								
			Р	=	bar	_		_	
1		a	f					е	С
		b	$LS_A$	=	bar	LS <sub>B</sub> =	b	ar	b
2	-	a	f					е	С
		b	$LS_A$	=	bar	LS <sub>B</sub> =	b	ar	b
3		а	f				_	е	С
		b	$LS_A$	=	bar	LS <sub>B</sub> =	b	ar	b
4		а	f					е	С
		b	$LS_A$	=	bar	LS <sub>B</sub> =	b	ar	b
5	Į.	а	f					е	С
		b	$LS_A$	=	bar	LS <sub>B</sub> =	b	ar	b
6		а	f					е	С
		b	$LS_A$	=	bar	LS <sub>B</sub> =	b	ar	b
7		а	f					е	С
		b	$LS_A$	=	bar	LS <sub>B</sub> =	b	ar	b
8		а	f					е	С
		b	$LS_A$	=	bar	LS <sub>B</sub> =	b	ar	b
9		а	f					е	С
		b	$LS_A$	=	bar	LS <sub>B</sub> =	b	ar	b
10		а	f					е	С
		b	$LS_A$	=	bar	LS <sub>B</sub> =	b	ar	b
11		а	f					е	С
_''		b	$LS_A$	=	bar	LS <sub>B</sub> =	b	ar	b
12	[	а	f					е	С
12		b	$LS_A$	=	bar	LS <sub>B</sub> =	b	ar	b
13	[	а	f					е	С
10		b	$LS_A$	=	bar	LS <sub>B</sub> =	b	ar	b
14	[	а	f					е	С
14		b	$LS_A$	=	bar	LS <sub>B</sub> =	b	ar	b
15	End section								
16	PVAS section								
17	7 Reserved for painting								

Comments	
Filled in by	Date

# Standard and option assembly

The PVG 32 valve group is assembled the way the module selection chart shows if the code number for PVM is written in field 'a', and the code number for PVMD, PVE or PVH in field 'c'.

The valve group is assembled so that the mechanical actuator is mounted on the opposite end of the basic module, if the code number for PVM is written in field 'c' of the order form and the code numbers for PVMD, PVE or PVH in field 'a'.







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